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Technical Report No. 4

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UNIQUENESS OF THE CONCENTRATED-LOAD PROBLEM IN THE LINEAR THEORY OF COUPLE-STRESS ELASTICITY

by

R. J. Hartranft

G. C. Sih

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TECHNICAL REPORT
NO. 4
ABERDEEN PROVINCIAL ENGINEERING
STEAP-TL

UNIQUENESS OF THE CONCENTRATED-LOAD PROBLEM IN THE LINEAR THEORY
OF COUPLE-STRESS ELASTICITY¹

by

R. J. Hartranft² and G. C. Sih³

In recent years, considerable attention has been focused on the linear theory of couple-stress elasticity. This Note is concerned with the development of certain conditions for uniqueness of solution in the couple-stress theory involving concentrated surface loads.

Because of the extensiveness of the literature on couple-stress problems, only those references which are relevant to the present investigation will be cited. The influence of couple-stresses on the stress distribution in a semi-infinite plane subjected to concentrated surface loads was studied by Muki and Sternberg [1]⁴, Tiwari [2], and Bert and Appl [3]. In [2,3], the conventional stresses were found to coincide with the Boussinesq solution of classical elasticity, while the couple-stresses were found to possess singularities of order r^{-2} , r being the radial distance measured from the point of application of the load. With the aid of Fourier transforms, Muki and Sternberg [1] solved the same problem but obtained results that disagree seriously with those in [2,3]. For the case of a concentrated load applied normal to the surface of a half-plane, one of the couple-stresses possessed merely a logarithmic singularity and the other remained finite at $r = 0$. In addition, the detailed structure of the singular terms of the conventional stresses is entirely different from that of the classical solution. The disagreement between the singular solutions in [1] and [2,3] could not be settled by the uniqueness theorem of Mindlin and Tiersten [4], since their theorem does not hold in the presence of discontinuous boundary loads. The need for a unique characterization of the concentrated-load problem in couple-stress elasticity is apparent.

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²Assistant Professor of Mechanics, Lehigh University, Bethlehem, Pa.

³Professor of Mechanics, Lehigh University, Bethlehem, Pa. Member ASME.

⁴Number in brackets designate References at end of Note.

Within the framework of the classical theory of elasticity, Sternberg and Eubanks [5] extended the classical uniqueness theorem to boundary-value problems with concentrated loads. They replaced the concentrated load by a system of statically equivalent surface tractions which are distributed continuously over a finite surface element around the concentrated-load point. The solution to the original problem is then defined as the limit of the solution to the distributed-loading problem, which is covered by the classical uniqueness theorem, as the surface element is shrunk to the load point. This limit-definition will also be adopted here in an effort to provide a unique formulation of the concentrated-load problem in the couple-stress theory of linear elasticity.

Stored Energy in Cosserat Medium

Before proceeding with the proof of uniqueness of solution in the presence of concentrated loads, the stored-energy expression will be cast into a convenient form. Under the conditions of plane strain [6], the energy density function for a Cosserat medium is

$$2W = \sigma_x \epsilon_x + \sigma_y \epsilon_y + (\tau_{xy} + \tau_{yx}) \epsilon_{xy} + \mu_x \kappa_x + \mu_y \kappa_y, \quad (1)$$

in which the curvatures κ_x, κ_y are proportional to the couple-stresses μ_x, μ_y ⁵:

$$\kappa_x = \frac{1}{4\eta} \mu_x, \quad \kappa_y = \frac{1}{4\eta} \mu_y. \quad (2)$$

The strain and stress relationships are

$$\begin{aligned} \epsilon_x &= \frac{1}{2G} [\sigma_x - \nu(\sigma_x + \sigma_y)] , & \epsilon_y &= \frac{1}{2G} [\sigma_y - \nu(\sigma_x + \sigma_y)] , \\ \epsilon_{xy} &= \frac{1}{4G} (\tau_{xy} + \tau_{yx}) . \end{aligned} \quad (3)$$

Substitution of eqs. (2) and (3) into (1) gives

$$2W = \lambda(\epsilon_x + \epsilon_y)^2 + 2G(\epsilon_x^2 + \epsilon_y^2 + 2\epsilon_{xy}^2) + 4\eta(\kappa_x^2 + \kappa_y^2), \quad (4)$$

⁵The constant η in eq. (2) corresponds to the modulus of curvature or bending, B , in Mindlin's paper [6].

where λ , G are the Lamé coefficients and they are related to Poisson's ratio ν as

$$\lambda = 2G\nu / (1-2\nu) .$$

In order that W in eq. (4) is positive definite, it is necessary and sufficient to require

$$\lambda > 0 , \quad G > 0 , \quad \eta > 0 .$$

Moreover, knowing that the strains are related to the displacements u_x , u_y by⁶

$$\epsilon_x = u_{x,x} , \quad \epsilon_y = u_{y,y} , \quad 2\epsilon_{xy} = u_{x,y} + u_{y,x} , \quad (5)$$

and the curvatures to the rotation, $2\omega_z = u_{y,x} - u_{x,y}$, by

$$\kappa_x = \omega_{z,x} , \quad \kappa_y = \omega_{z,y} \quad (6)$$

eq. (1) or (4) may also be written in the form

$$2W = (\sigma_x u_x + \tau_{xy} u_y + \mu_{xz} \omega_z)_{,x} + (\sigma_y u_y + \tau_{yx} u_x + \mu_{yz} \omega_z)_{,y} . \quad (7)$$

In deriving eq. (7), use has been made of the equations of static equilibrium

$$\sigma_{x,x} + \tau_{yx,y} = 0 , \quad \tau_{xy,x} + \sigma_{y,y} = 0 , \quad \tau_{xy} - \tau_{yx} + \mu_{x,x} + \mu_{y,y} = 0 . \quad (8)$$

Now, the total energy stored in the Cosserat medium may be obtained by integrating eq. (7) and applying the divergence theorem. The result is

$$2 \iint_R W \, dA = \int_L [(\sigma_x u_x + \tau_{xy} u_y + \mu_{xz} \omega_z) dy - (\sigma_y u_y + \tau_{yx} u_x + \mu_{yz} \omega_z) dx] . \quad (9)$$

Expressing all quantities in eq. (9) in the directions normal and tangent to the boundary L , eq. (9) becomes

$$2 \iint_R W \, dA = \int_L (\sigma_n u_n + \tau_{ns} u_s + \mu_n \omega_z) \, ds . \quad (10)$$

Uniqueness Theorem for Concentrated Loads

Based on the positive definiteness of W and eq. (10), the following

⁶ A comma is used to indicate differentiation such as $u_{x,x} = \partial u_x / \partial x$, etc.

uniqueness theorem in the couple-stress theory of elasticity may be established:

Let $\sigma_x^{(1)}, \sigma_y^{(1)}, \dots, \omega_z^{(1)}$ and $\sigma_x^{(2)}, \sigma_y^{(2)}, \dots, \omega_z^{(2)}$ be two possible solutions which are continuous and have piecewise continuous first partial derivatives in an open region containing R and its boundary L . Then the difference solution $\Delta\sigma_x = \sigma_x^{(1)} - \sigma_x^{(2)}$, etc. vanishes if and only if

$$\int_L (\Delta\sigma_n \Delta u_n + \Delta\tau_{ns} \Delta u_s + \Delta\mu_n \Delta \omega_z) ds = 0 \quad (11)$$

The aforementioned theorem can be applied to problems involving singular loads if the points at which stress discontinuities occur are cut out from the region R . The original problem is then recovered by letting the size of the cut vanish. For definiteness sake, let a point O on the boundary L be subjected to concentrated forces p, q and a couple m as shown in Fig. 1(a). Now, consider a small semi-circular indentation of radius ρ removed from R and subject it to a system of finite stresses which are statically equivalent to p, q and m as

$$p = \int_{\ell} [\sigma_n^{(i)} \cos\theta - \tau_{ns}^{(i)} \sin\theta] \rho d\theta, \quad q = \int_{\ell} [\sigma_n^{(i)} \sin\theta + \tau_{ns}^{(i)} \cos\theta] \rho d\theta, \quad (12)$$

$$m = \int_{\ell} [\mu_n^{(i)}] \rho d\theta$$

where $i = 1, 2$ and ℓ represents the interval $-\pi/2 \leq \theta \leq \pi/2$. The region R_1 in Fig. 1(b) is defined such that $R_1 \rightarrow R$ when ρ approaches zero. The expressions for p, q and m in eq. (12) are required to be integrable in the limit as $\rho \rightarrow 0$. Thus, the order of the stress singularities for $\sigma_n^{(i)}, \tau_{ns}^{(i)}$ and $\mu_n^{(i)}$ can at most be r^{-1} . Taking the difference of the two possible stress states denoted by

$$r^{-1}f_1(\theta) = \Delta\sigma_n \cos\theta - \Delta\tau_{ns} \sin\theta, \quad r^{-1}f_2(\theta) = \Delta\sigma_n \sin\theta + \Delta\tau_{ns} \cos\theta, \quad r^{-1}f_3(\theta) = \Delta\mu_n,$$

eq. (12) reduces to

$$\int_{\ell} f_i(\theta) d\theta = 0, \quad i = 1, 2, 3 \quad (13)$$

Using eq. (13) and applying eq. (11) to the problem illustrated in Fig. 1(b) render the condition for uniqueness of solution⁷:

$$\int_{\ell} [f_1(\theta) g_1(\rho, \theta) + \dots + f_3(\theta) g_3(\rho, \theta)] d\theta = 0, \quad (14)$$

in which $g_i(\rho, \theta)$ are continuous functions of ρ, θ and are given by

$$g_1(\rho, \theta) = \Delta u_n \cos \theta - \Delta u_s \sin \theta, \quad g_2(\rho, \theta) = \Delta u_n \sin \theta + \Delta u_s \cos \theta, \quad g_3(\rho, \theta) = \Delta \omega_z.$$

Mean-Value Theorem

To establish eq. (14), the generalized first mean-value theorem of the integral calculus will be employed:

Let $f(x)$ and $g(x)$ be two continuous functions in the interval $a \leq x \leq b$, where $f(x) \geq 0$. There exists a number α intermediate between a and b such that

$$\int_a^b f(x) g(x) dx = g(\alpha) \int_a^b f(x) dx$$

Further, if L represents the union of disjoint intervals on each of which $f(x)$ is always positive (or negative), then the extension of the above theorem is

$$\int_{\ell} f(x) g(x) dx \leq g(\alpha) \int_{\ell} f(x) dx, \quad \alpha \in \ell \quad (15)$$

The notation

$$I_i(\rho) = \int_{\ell} f_i(\theta) g_i(\rho, \theta) d\theta, \quad i = 1, 2, 3 \text{ (no sum on } i) \quad (16)$$

which stands for a typical term of eq. (14), will be adopted. Therefore, it suffices to establish the uniqueness of solution by showing that $I_i(\rho) \rightarrow 0$ as $\rho \rightarrow 0$. Letting $\ell = \ell_1 + \ell_2$ with the requirements that

⁷The usual boundary conditions are satisfied on $L-\ell$ so that

$$\int_{L-\ell} [\Delta \sigma_n \Delta u_n + \dots + \Delta \mu_n \Delta \omega_z] ds = 0.$$

$$f_i(\theta) \geq 0 \quad \text{on } \ell_1, \text{ and } f_i(\theta) \leq 0 \quad \text{on } \ell_2$$

eq. (13) yields

$$\int_{\ell_1} f_i(\theta) d\theta = - \int_{\ell_2} f_i(\theta) d\theta = K > 0 \quad . \quad (17)$$

Making use of eqs. (15) and (17), eq. (16) may be put into the form

$$\begin{aligned} I_i(\rho) &= \int_{\ell_1} f_i(\theta) g_i(\rho, \theta) d\theta + \int_{\ell_2} f_i(\theta) g_i(\rho, \theta) d\theta \\ &\leq K[g_i(\rho, \theta_1) - g_i(\rho, \theta_2)] \end{aligned} \quad (18)$$

Since $g_i(\rho, \theta)$ is continuous on ℓ , there exists a $\delta_1 > 0$ such that when the distance between the points (ρ, θ_1) and (ρ, θ_2) is less than δ_1 , the condition

$$|g_i(\rho, \theta_1) - g_i(\rho, \theta_2)| < \frac{\epsilon}{K}$$

holds for every positive number ϵ . Hence, there is a $\delta > 0$ such that whenever $\rho < \delta$ ⁸,

$$|I_i(\rho)| < \epsilon \quad \text{for } \epsilon > 0$$

and thus

$$\lim_{\rho \rightarrow 0} I_i(\rho) = 0 \quad .$$

This completes the proof of the uniqueness theorem for concentrated-load problems in the linear couple-stress theory of elasticity.

Concluding Remarks

The results of the present investigation provide the following conditions for uniqueness:

(1) Conventional- and couple-stresses must be continuous and have piecewise continuous first partial derivatives at every point of the medium except, perhaps, at the point where the concentrated loads are applied. The same conditions must be satisfied by the displacements and rotation. Geometric discontinuities are to be excluded.

⁸For this problem, $\delta = 1/2 (\delta_1)$.

(2) All quantities such as stresses, displacements, etc. must vanish as $r \rightarrow \infty$ if the boundary extends to infinity.

(3) The conventional- and couple-stress singularities can at most be $O(r^{-1})$, where r is the radial distance measured from the point of application of the concentrated loads.

(4) The stress system on a semi-circular cut about $r = 0$ must be statically equivalent to the applied loads.

(5) At other points of the boundary, the usual boundary conditions such as σ_n , τ_{ns} and μ_n must satisfy their prescribed values.

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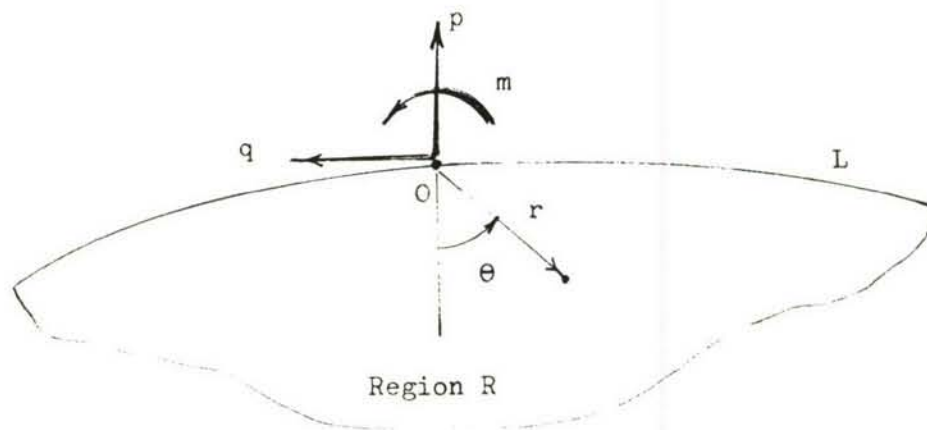


Fig. 1(a) - Concentrated forces and couple.

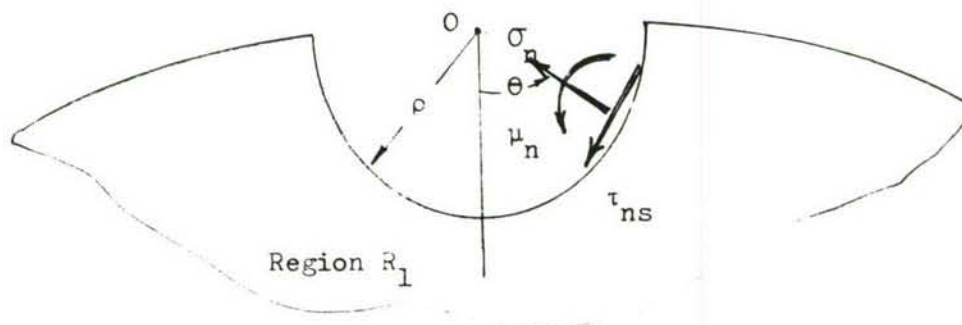


Fig. 1(b) - Equivalent distributed loads.

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Prof. Tsuyoshi Hayashi
Department of Aeronautics
Faculty of Engineering
University of Tokyo
BUNKYO-KU
Tokyo, Japan

Prof. R. J. H. Bollard
Chairman, Aeronautical Engr. Dept.
207 Guggenheim Hall
University of Washington
Seattle, Washington 98105

Prof. Albert S. Kobayashi
Dept. of Mechanical Engr.
University of Washington
Seattle, Washington 98105

Officer-in-Charge
Post Graduate School for Naval Off.
Webb Institute of Naval Arch.
Crescent Beach Road, Glen Cove
Long Island, New York 11542

Industry and Research Institutes

Mr. K. W. Bills, Jr.
Dept. 4722, Bldg. 0525
Aerojet-General Corporation
P. O. Box 1947
Sacramento, California 95809

Dr. James H. Wiegand
Senior Dept. 4720, Bldg. 0525
Ballistics & Mech. Properties Lab.
Aerojet-General Corporation
P. O. Box 1947
Sacramento, California 95809

Dr. John Zickel
Dept. 4650, Bldg. 0227
Aerojet-General Corporation
P. O. Box 1947
Sacramento, California 95809

Mr. J. S. Wise
Aerospace Corporation
P. O. Box 1308
San Bernardino, California 92402

Dr. Vito Salerno
Applied Technology Assoc., Inc.
29 Church Street
Ramsey, New Jersey 07446

Library Services Department
Report Section, Bldg. 14-14
Argonne National Laboratory
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Argonne, Illinois 60440

Dr. E. M. Kervin
Bolt, Beranek, & Newman, Inc.
50 Moulton Street
Cambridge, Massachusetts 02138

Dr. M. C. Junger
Cambridge Acoustical Associates
129 Mount Auburn Street
Cambridge, Massachusetts 02138

Dr. F. R. Schwarzl
Central Laboratory T.N.O.
134 Julianalaan
Delft, Holland

Universities (cont'd)

Prof. H. T. Corten
University of Illinois
Urbana, Illinois 61803

Prof. W. J. Hall
Department of Civil Engr.
University of Illinois
Urbana, Illinois 61803

Prof. N. M. Newmark
Dept. of Civil Engineering
University of Illinois
Urbana, Illinois 61803

Dr. W. H. Avery
Applied Physics Laboratory
Johns Hopkins University
8621 Georgia Avenue
Silver Spring, Maryland 20910

Prof. J. B. Tiedemann
Dept. of Aero. Engr. & Arch.
University of Kansas
Lawrence, Kansas 66045

Prof. S. Taira
Department of Engineering
Kyoto University
Kyoto, Japan

Prof. E. Reissner
Dept. of Mathematics
Massachusetts Inst. of Tech.
Cambridge, Massachusetts 02139

Library (Code 0384)
U. S. Naval Postgraduate School
Monterey, California 93940

Dr. Joseph Marin
Prof. of Materials Science
Dept. of Materials Sc. & Chem.
U. S. Naval Postgraduate School
Monterey, California 93940

Prof. E. L. Reiss
Courant Inst. of Math. Sciences
New York University
4 Washington Place
New York, New York 10003

Dr. Francis Cozzarelli
Div. of Interdisciplinary
Studies and Research
School of Engineering
State Univ. of N.Y. at Buffalo
Buffalo, New York 14214

Dr. George Herrmann
The Technological Institute
Northwestern University
Evanston, Illinois 60201

Director, Ordnance Research Lab.
The Pennsylvania State University
P. O. Box 30
State College, Pennsylvania 16801

Prof. Eugen J. Skudrzyk
Department of Physics
Ordnance Research Lab.
The Pennsylvania State University
P. O. Box 30
State College, Pennsylvania 16801

Dean Oscar Baguio
Assoc. of Structural Engr.
of the Philippines
University of Philippines
Manila, Philippines

Prof. J. Kempner
Dept. of Aero. Engr. & Applied Mech.
Polytechnic Institute of Brooklyn
333 Jay Street
Brooklyn, New York 11201

Prof. J. Klossner
Polytechnic Institute of Brooklyn
333 Jay Street
Brooklyn, New York 11201

Prof. F. R. Eirich
Polytechnic Institute of Brooklyn
333 Jay Street
Brooklyn, New York 11201

Prof. A. C. Eringen
School of Aero., Astro. & Engr. Sc.
Purdue University
Lafayette, Indiana 47907

Industry & Research Inst. (cont'd.)

Mr. Ronald D. Brown
Applied Physics Laboratory
Chemical Propulsion Agency
8621 Georgia Avenue
Silver Spring, Maryland 20910

Research and Development
Electric Boat Division
General Dynamics Corporation
Groton, Connecticut 06340

Supervisor of Shipbuilding, USN,
and Naval Insp. of Ordnance
Electric Boat Division
General Dynamics Corporation
Groton, Connecticut 06340

Dr. L. H. Chen
Basic Engineering
Electric Boat Division
General Dynamics Corporation
Groton, Connecticut 06340

Mr. Ross H. Petty
Technical Librarian
Allagany Ballistics Lab.
Hercules Powder Company
P. O. Box 210
Cumberland, Maryland 21501

Dr. J. H. Thacher
Allagany Ballistics Laboratory
Hercules Powder Company
Cumberland, Maryland 21501

Dr. Joshua E. Greenspon
J. H. Engr. Research Associates
3811 Menlo Drive
Baltimore, Maryland 21215

Mr. R. F. Landel
Jet Propulsion Laboratory
4800 Oak Grove Drive
Pasadena, California 91103

Industry & Research Inst. (cont'd.)

Dr. R. C. DeHart
Southwest Research Institute
8500 Culebra Road
San Antonio, Texas 78206

Dr. Thor Smith
Stanford Research Institute
Menlo Park, California 94025

Mr. J. Edmund Fitzgerald
Director, Research & Engr.
Lockheed Propulsion Company
P. O. Box 111
Redlands, California 92374

Library
Newport News Shipbuilding &
Dry Dock Company
Newport News, Virginia 23607

Mr. E. A. Alexander
Rocketdyne Division
North American Aviation, Inc.
6633 Canoga Avenue
Canoga Park, California 91304

Mr. Cesar P. Nguind
Deputy Commissioner
Philippine Atomic Energy Commission
Manila, Philippines

Mr. S. C. Britton
Solid Rocket Division
Rocketdyne
P. O. Box 548
McGregor, Texas 76657

Dr. A. J. Ignatowski
Redstone Arsenal Research Div.
Rohm & Haas Company
Huntsville, Alabama 35807

Dr. M. L. Merritt
Division 5412
Sandia Corporation
Sandia Base
Albuquerque, New Mexico 87115

Director
Ship Research Institute
Ministry of Transportation
700, SHINKAWA
Mitaka
Tokyo, JAPAN

Dr. H. N. Abrahamson
Southwest Research Institute
8500 Culebra Road
San Antonio, Texas 78206

Dr. M. L. Baron
Paul Weidinger, Consulting Engr.
777 Third Ave. - 22nd Floor
New York, New York 10017